#### ECE171A: Linear Control System Theory Lecture 12: Stability Margins

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Stability Margins from a Nyquist Plot



• Consider an open-loop transfer function:  $G(s) = K \frac{\prod_{i=1}^{m} (s - z_i)}{\prod_{i=1}^{n} (s - p_i)}$ 

• Changing K scales the magnitude of every point on the contour G(C)

Increasing K pushes all points on the Nyquist plot of G(s) further away from the origin

### Stability Margins from a Nyquist Plot: Example

• Draw a Nyquist plot for  $G(s) = \frac{K}{s(s+1)(s+10)}$ 



The closed-loop system is stable for small K and unstable for large K

# Gain Margin

- Gain Margin:
  - the factor by which the open-loop gain can be increased before a stable closed-loop system becomes unstable
  - the factor by which the open-loop gain should be decreased until an unstable system becomes stable
- On a Nyquist plot, the gain margin is the inverse of the distance to the first point where G(C) crosses the real axis



# Phase Margin

- Phase Margin:
  - the amount by which the open-loop phase can be decreased before a stable closed-loop system becomes unstable
  - the amount by which the open-loop phase should be increased before an unstable system becomes stable
- ► On a Nyquist plot, the phase margin is the smallest angle on the unit circle between −1 and G(C)



Algebraic Definitions of Gain Margin and Phase Margin

Phase-Crossover frequency: the frequency ω<sub>p</sub> at which G(jω) crosses the real axis:

$$/G(j\omega_p) = -\pi$$

• Gain Margin: the inverse of the open-loop gain at  $\omega_p$ :

$$GM = 20 \log \frac{1}{|G(j\omega_p)|} = -20 \log |G(j\omega_p)| \text{ dB}$$

Gain-Crossover frequency: the frequency ω<sub>g</sub> at which G(jω) crosses the unit circle:

$$20\log|G(j\omega_g)|=0 \text{ dB}$$

Phase Margin: the amount by which the open-loop phase at ω<sub>g</sub> exceeds -π:

$$PM = \underline{/G(j\omega_g)} + \pi$$

### Gain Margin and Phase Margin

- For a stable minimum-phase system both GM and PM are positive. Larger gains mean larger relative stability.
- When  $\omega_g = \omega_p = \omega_*$ , there are closed-loop poles on the imaginary axis:

$$|G(j\omega_*)| = 1, \qquad \underline{/G(j\omega_*)} = -\pi \qquad \Rightarrow \qquad 1 + G(j\omega_*) = 0$$

- ► A Bode plot or a Magnitude-Phase plot provides  $|G(j\omega)|$  and  $/G(j\omega)$
- Hence, phase-crossover frequency, gain-crossover frequency, gain margin, phase margin can all be seen on a Bode plot or a Magnitude-Phase plot
- Caution: the Bode plot or magnitude-phase plot interpretation of gain and phase margins to determine stability can be incorrect if the system is non-minimum phase or has delays. Only the Nyquist stability criterion should be used to determine stability.

### Gain Margin and Phase Margin on a Magnitude-Phase Plot

• Magnitude-phase plot of  $G_1(s) = \frac{1}{s(s+1)(s/5+1)}$  and  $G_2(s) = \frac{1}{s(s+1)^2}$ 



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▶ Bode plot of  $G(s) = \frac{K}{s(s+1)(s/100+1)}$  with K = 1



- If K > 0, it has no effect on the phase and shifts the magnitude up or down by 20 log K. This changes the gain-crossover frequency but not the phase-crossover frequency.
- ▶ Some closed-loop poles lie on the imaginary axis when  $\omega_g = \omega_p$
- Choose  $K \approx 100$  to shift the magnitude up by  $\sim 40$  dB, making  $\omega_g \approx \omega_p$
- The imaginary axis crossing can be determined from the Bode plot but we do not know if we are going from stability to instability or vice versa
- Assuming that the system is stable initially (can only be verified by the Nyquist or the Routh-Hurwitz stability criterion), we expect the region of stability to be 0 < K < 100

• Root locus of  $G(s) = \frac{\kappa}{s(s+1)(s/100+1)}$ 



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Use Routh-Hurwitz to verify the region of stability for:

$$T(s) = \frac{G(s)}{1 + G(s)} = \frac{K}{s(s+1)(s/100+1) + K} = \frac{100K}{s^3 + 101s^2 + 100s + 100K}$$

- Characteristic polynomial  $a(s) = s^3 + 101s^2 + 100s + 100K$
- The Routh table is:

<i>s</i> <sup>3</sup>	1	100
<i>s</i> <sup>2</sup>	101	100 <i>K</i>
$s^1$	$100 - \frac{100K}{101}$	0
<i>s</i> <sup>0</sup>	100 <i>K</i>	0

- Stability region: 0 < K < 101
- Auxiliary polynomial roots for K = 101:

 $A(s) = 101(s^2 + 100) \qquad \Rightarrow \qquad s = \pm j10$ 

### Stability Margins: Example 1

• What are the gain margin and phase margin of  $G(s) = \frac{1}{s(s+1)^2}$ ?



### Stability Margins: Example 2

• What are the gain margin and phase margin of  $G(s) = \frac{(s+1)}{s^2(s/10+1)}$ ?



Stability Margins: Example 2 • Root locus of  $G(s) = \frac{(s+1)}{s^2(s/10+1)}$ 



# Stability Margins: Example 2

- What are the gain margin and phase margin of  $G(s) = \frac{K(s+1)}{s^2(s/10+1)}$ ?
- ▶ The gain margin is  $\infty$  since the phase hits  $-180^\circ$  at  $\omega_p = \infty$
- As K → ∞, the gain-crossover frequency ω<sub>g</sub> moves to the right and the phase margin decreases
- ▶ Root locus: a set of poles move vertically in the plane ( $\zeta$  decreases) as  $K \to \infty$
- $\blacktriangleright$  There seems to be a relationship between phase margin and the damping ratio  $\zeta$
- We will analyze a second-order system to determine this and, more generally, the relationship between frequency response and transient response

## Frequency Domain Performance Specifications



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Consider a second-order system:

$$T(s) = \frac{G(s)}{1+G(s)} = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} = \frac{1}{\frac{s^2}{\omega_n^2} + 2\zeta\frac{s}{\omega_n} + 1}$$

How does the closed-loop frequency response T(jω) relate to the time-domain transient response (rise time, overshoot, settling time)?

### Frequency Response of a Second-order System

• Bode plot of 
$$T(s) = \frac{1}{\frac{s^2}{\omega_s^2} + 2\zeta \frac{s}{\omega_n} + 1}$$



- The damping ratio  $\zeta$  is related to max<sub> $\omega$ </sub>  $|T(j\omega)|$
- The rise time t<sub>r</sub> is related to the bandwidth ω<sub>b</sub>, which measures the frequency range over which the system tracks an input signal well
- The natural frequency  $\omega_n$  is related to the bandwidth  $\omega_b$

### Frequency Domain Performance Specifications

- ▶ Low-frequency (DC) gain: the magnitude of the transfer function  $|T(j\omega)|$  for low frequencies  $\omega \rightarrow 0$
- Bandwidth: the frequency ω<sub>b</sub> at which the magnitude of the transfer function drops to 3 dB below the DC gain:

$$|T(j\omega_b)| = \frac{1}{\sqrt{2}}|T(0)|$$

Resonant frequency: the frequency at which the transfer function magnitude is maximized

$$\omega_r = rg\max_{\omega} |T(j\omega)|$$

Resonant peak: the maximum value of the transfer function magnitude:

$$M_r = |T(j\omega_r)|$$

### Frequency Domain Performance Specifications



#### Frequency Response of a Second-order System

Consider a second-order system:

$$T(s) = \frac{\omega_n^2}{s^2 + 2\zeta\omega_n s + \omega_n^2} = \frac{1}{\frac{s^2}{\omega_n^2} + 2\zeta\frac{s}{\omega_n} + 1}$$

• Transfer function magnitude at  $s = j\omega$ :

$$|T(j\omega)| = \frac{1}{|-\frac{\omega^2}{\omega_n^2} + 2\zeta \frac{\omega}{\omega_n} j + 1|} = \frac{1}{\sqrt{\left(1 - \left(\frac{\omega}{\omega_n}\right)^2\right)^2 + 4\zeta^2 \left(\frac{\omega}{\omega_n}\right)^2}}$$

• Transfer function phase at  $s = j\omega$ :

$$\underline{/T(j\omega)} = \underline{/\frac{1}{-\left(\frac{\omega}{\omega_n}\right)^2 + 2\zeta\left(\frac{\omega}{\omega_n}\right)j+1}} = -\arctan\left(\frac{2\zeta\left(\frac{\omega}{\omega_n}\right)}{1-\left(\frac{\omega}{\omega_n}\right)^2}\right)$$

Resonant Frequency of a Second-order System

Resonant frequency:

$$\frac{d|T(j\omega)|}{d\omega} = 0 \qquad \Rightarrow \qquad \omega_r = 0 \quad \mathsf{OR} \quad \omega_r = \omega_n \sqrt{1 - 2\zeta^2}$$

Resonant peak:
Case 1:  $\zeta \leq \frac{1}{\sqrt{2}}$ :  $\omega_r = \omega_n \sqrt{1 - 2\zeta^2} \qquad M_r = \frac{1}{2\zeta\sqrt{1 - \zeta^2}}$ Case 2:  $\zeta > \frac{1}{\sqrt{2}}$ :  $\omega_r = 0 \qquad M_r = 1$  Resonant Frequency of a Second-order System

• Plot of 
$$M_r = \frac{1}{2\zeta\sqrt{1-\zeta^2}}$$
 and  $\frac{\omega_r}{\omega_n} = \sqrt{1-2\zeta^2}$  as a function of  $\zeta$ 

- The resonant peak M<sub>r</sub> is related to the percent overshoot via ζ
- Example:
  - ► The resonant peak of the closed-loop system should be less than 1.75 (≈ 5 dB)
  - Equivalent to ζ should be greater than 0.3
  - Equivalent to p.o should be less than 37%



# Bandwidth of a Second-order System

- Bandwidth: the low frequency range (0, ω<sub>b</sub>) over which the closed-loop system tracks an input signal well
- ▶ Bandwidth:  $\omega$  such that  $|T(j\omega)| = \frac{1}{\sqrt{2}} |T(0)|$ . Let  $u = \omega_b/\omega_n$ :

$$u^{4} + 2(\zeta^{2} - 1)u^{2} + 1 = 2 \quad \Rightarrow \quad u^{2} = (1 - 2\zeta^{2}) \pm \sqrt{4\zeta^{4} - 4\zeta^{2} + 2}$$
$$\omega_{b} = \omega_{n}\sqrt{(1 - 2\zeta^{2}) + \sqrt{4\zeta^{4} - 4\zeta^{2} + 2}}$$

- Bandwidth ω<sub>b</sub> and rise time t<sub>r</sub> ≈ 2.16ζ+0.6/ω<sub>n</sub> are inversely proportional:
   If ω<sub>n</sub> ↑, then ω<sub>b</sub> ↑ and t<sub>r</sub> ↓
   If ζ ↑, then ω<sub>b</sub> ↓ and t<sub>r</sub> ↑
- Adding a zero to G(s) increases the bandwidth of the closed-loop transfer function T(s)
- Adding a pole to G(s) decreases the bandwidth of the closed-loop transfer function T(s)

#### Stability Margins of a Second-order System

• Bode plot of 
$$G(s) = \frac{\omega_n^2}{s(s+2\zeta\omega_n)}$$



#### Stability Margins of a Second-order System

The phase plot of G(s) shows that the phase-crossover frequency is:

$$\omega_p = \infty$$

The gain margin is:

$$GM = \infty$$

Set  $|G(j\omega)|$  to 1 to obtain the gain-crossover frequency  $\omega_g$ :

$$1 = |G(j\omega_g)| = \frac{\omega_n^2}{|j\omega_g||j\omega_g + 2\zeta\omega_n|} = \frac{\omega_n^2}{\omega_g\sqrt{4\zeta^2\omega_n^2 + \omega_g^2}}$$

The gain-crossover frequency is:

$$\omega_g = \omega_n \sqrt{\sqrt{1 + 4\zeta^4} - 2\zeta^2}$$

The phase margin is:

$$\mathsf{PM} = \underline{/G(j\omega_g)} + \pi = \tan^{-1}\left(\frac{2\zeta}{\sqrt{\sqrt{1+4\zeta^4}-2\zeta^2}}\right)$$

# Phase Margin of a Second-order System

- The phase margin of a second-order system is a function of  $\zeta$  but not  $\omega_n$
- The relationship between PM and ζ can be approximated fairly well by a straight line for small values of ζ



# Phase Margin of a Second-order System

For 0 ≤ ζ ≤ 0.7, the phase margin *PM* (in degrees) and the damping ration ζ of a second-order system are related by:

 $PM \approx 100\zeta$ 

- The relationship between ζ and PM can be used to design control systems in the frequency domain meeting time-domain specifications
- Poles that are ignored in a dominant-pole-pair approximation contribute phase lag so it is important to keep a large phase margin
- For 0.2 ≤ ζ ≤ 0.8, the gain-crossover frequency ω<sub>g</sub> of G(s) is related to the closed-loop system bandwidth ω<sub>b</sub>:

$$\omega_b \approx 1.8 \omega_g$$

### Frequency Domain Control Design

- ▶ To obtain fast transient response we want large  $\omega_g$  since  $\omega_b \uparrow$ ,  $t_r \downarrow$
- To obtain lower steady-state error, we may increase the gain K, which increases ω<sub>g</sub>
- Increasing  $\omega_g$ , however, decreases the phase margin
- As the phase margin decreases, the system becomes less stable and might exhibit oscillatory behavior
- We must consider more complicated controllers than a simple proportional controller K to obtain a good phase margin, a good gain-crossover frequency, and good steady state tracking

Frequency Domain Performance Specifications



Unity-feedback control system with open-loop transfer function:

$$G(s) = K rac{\prod_{i=1}^m (s-z_i)}{\prod_{i=1}^n (s-p_i)}$$

- How can the closed-loop frequency-domain performance specifications (resonant peak M<sub>r</sub>, resonant frequency ω<sub>r</sub>, bandwidth ω<sub>b</sub>) be related to the open-loop frequency response?
- How can the gain K be adjusted to meet frequency-domain performance specifications?

# Closed-loop Transfer Function Magnitude

Closed-loop transfer function:

$$T(s) = \frac{Y(s)}{R(s)} = \frac{G(s)}{1 + G(s)}$$

Closed-loop transfer function magnitude:

$$M(s) = |T(s)| = \frac{|G(s)|}{|1 + G(s)|}$$

Obtain M(s) as a function of the real and imaginary parts of G(s) = x(s) + jy(s):

$$M = \frac{\sqrt{x^2 + y^2}}{\sqrt{(1 + x)^2 + y^2}}$$

This equation turns out to be a circle on a Nyquist plot

#### Constant Magnitude Circles

Relationship between the magnitude of the closed-loop transfer function M and the real part x and imaginary part y of the open-loop transfer function:

$$M^2(1+x)^2 + M^2y^2 = x^2 + y^2$$
  
 $M^2 = (1-M^2)x^2 - 2M^2x + (1-M^2)y^2$ 

Assume  $M \neq 1$  and divide both sides by  $(1 - M^2)$ :

$$x^2 - 2\frac{M^2}{1 - M^2}x + y^2 = \frac{M^2}{1 - M^2}$$

Add  $M^4/(1-M^2)^2$  to both sides to complete the square for x:

$$\left(x - \frac{M^2}{1 - M^2}\right)^2 + y^2 = \frac{M^2}{(1 - M^2)^2}$$

#### Constant Magnitude Circles

 M circle: a circle of constant closed-loop transfer function magnitude on a polar/Nyquist plot:

$$\left(x - \frac{M^2}{1 - M^2}\right)^2 + y^2 = \frac{M^2}{(1 - M^2)^2}$$

• An *M* circle is centered at  $\left(\frac{M^2}{1-M^2}, 0\right)$  with radius  $\frac{M}{|(1-M^2)|}$ 

- ▶ As  $M \to \infty$ , the M circle is centered at (-1,0) with radius 0
- For 1 < M < ∞, the M circle center moves to the left of (-1,0), while the radius increases
- As  $M \rightarrow 0$ , the M circle is centered at (0,0) with radius 0
- ► For 0 < M < 1, the M circle center moves to the right of (0,0), while the radius increases
- ▶ At M = 1, we get a degenerate circle at  $(\pm \infty, 0)$  with radius  $\infty$

Constant Magnitude Circles on a Nyquist Plot

- Nyquist plot for  $G(s) = \frac{4(s/2+1)}{s(2s+1)(1+0.4(s/8)+(s/8)^2)}$
- If the frequencies ω along the polar plot of G(s) are available, we can construct a closed-loop Bode plot using the M circles



#### **Constant Phase Circles**

▶ *N* circle: a circle of constant  $N = \tan T(s)$  on a polar/Nyquist plot:

$$\left(x+\frac{1}{2}\right)^2 + \left(y-\frac{1}{2N}\right)^2 = \frac{1}{4}\left(1+\frac{1}{N^2}\right)$$

An N circle is centered at (-0.5, 0.5/N) with radius  $0.5\sqrt{1+1/N^2}$ 

> N circles are orthogonal to M circles, i.e., intersect at 90 $^{\circ}$ 



# Frequency Domain Performance Specifications

Given the frequency response of an open-loop transfer function G(s), we can verify stability and frequency domain performance metrics

#### Stability:

- Determine using the Nyquist criterion
- What if K < 0? Rotate the Nyquist plot clockwise by 180°.
- Gain margin GM and phase margin PM:
  - Can be obtained from a Nyquist plot
  - Even easier to determine on a Bode plot or magnitude-phase plot

**•** Resonant peak  $M_r$ , resonant frequency  $\omega_r$ , and bandwidth  $\omega_b$ :

Use the *M* circles on a Nyquist plot

#### **Open-loop Bode Plot**

• Open-loop Bode plot for  $G(s) = \frac{4(s/2+1)}{s(2s+1)(1+0.4(s/8)+(s/8)^2)}$ 



#### Nyquist Plot

• Nyquist plot for  $G(s) = \frac{4(s/2+1)}{s(2s+1)(1+0.4(s/8)+(s/8)^2)}$ 



#### Closed-loop Bode Plot

• Closed-loop Bode plot for  $G(s) = \frac{4(s/2+1)}{s(2s+1)(1+0.4(s/8)+(s/8)^2)}$ 



### Frequency Domain Control Design

- How should K be adjusted to meet desired closed-loop frequency domain specifications?
  - It is difficult to determine how much to change K to meet a resonant peak specification on a Nyquist plot
  - It is difficult to tell where the Nyquist plot would become tangent to the desired *M* circle
- Nathaniel Nichols proposed to transform the M and N circles from a Nyquist plot to a magnitude-phase plot
- > On a magnitude-phase plot, the M and N contours are no longer circles
- If K changes, a magnitude-phase plot only moves up or down, which is much easier to interpret that the change of the shape on a Nyquist plot

# Nichols Chart

Nichols chart: a

magnitude-phase plot with overlaid M and N contours of constant closed-loop transfer-function magnitude and phase

- The gain margin and phase margin can be obtained
- The resonant peak and bandwidth can be obtained
- A change in the gain K moves the system response up or down and can be used to meet closed-loop frequency domain specifications



## Nichols Chart

• Nichols plot for  $G(s) = \frac{4(s/2+1)}{s(2s+1)(1+0.4(s/8)+(s/8)^2)}$ 

